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APPLICATION NO.	FILING DATE	FIRST NAMED INVENTOR	ATTORNEY DOCKET NO.	CONFIRMATION NO.
09/707,865	11/07/2000	Bengt Ebbeson	30882US1	1443
116	7590	05/01/2007	EXAMINER	
PEARNE & GORDON LLP			LEO, LEONARD R	
1801 EAST 9TH STREET				
SUITE 1200			ART UNIT	PAPER NUMBER
CLEVELAND, OH 44114-3108			3744	
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Please find below and/or attached an Office communication concerning this application or proceeding.

The time period for reply, if any, is set in the attached communication.

Office Action Summary	Application No.	Applicant(s)	
	09/707,865	EBBESON, BENGT	
	Examiner	Art Unit	
	Leonard R. Leo	3744	

-- The MAILING DATE of this communication appears on the cover sheet with the correspondence address --
Period for Reply

A SHORTENED STATUTORY PERIOD FOR REPLY IS SET TO EXPIRE 3 MONTH(S) OR THIRTY (30) DAYS, WHICHEVER IS LONGER, FROM THE MAILING DATE OF THIS COMMUNICATION.

- Extensions of time may be available under the provisions of 37 CFR 1.136(a). In no event, however, may a reply be timely filed after SIX (6) MONTHS from the mailing date of this communication.
 - If NO period for reply is specified above, the maximum statutory period will apply and will expire SIX (6) MONTHS from the mailing date of this communication.
 - Failure to reply within the set or extended period for reply will, by statute, cause the application to become ABANDONED (35 U.S.C. § 133).
- Any reply received by the Office later than three months after the mailing date of this communication, even if timely filed, may reduce any earned patent term adjustment. See 37 CFR 1.704(b).

Status

- 1) Responsive to communication(s) filed on 30 August 2004.
- 2a) This action is FINAL. 2b) This action is non-final.
- 3) Since this application is in condition for allowance except for formal matters, prosecution as to the merits is closed in accordance with the practice under *Ex parte Quayle*, 1935 C.D. 11, 453 O.G. 213.

Disposition of Claims

- 4) Claim(s) 1,5-19,45 and 46 is/are pending in the application.
- 4a) Of the above claim(s) 8-12,15 and 17-19 is/are withdrawn from consideration.
- 5) Claim(s) _____ is/are allowed.
- 6) Claim(s) 1,5-7,13,14,16,45 and 46 is/are rejected.
- 7) Claim(s) _____ is/are objected to.
- 8) Claim(s) _____ are subject to restriction and/or election requirement.

Application Papers

- 9) The specification is objected to by the Examiner.
- 10) The drawing(s) filed on _____ is/are: a) accepted or b) objected to by the Examiner.
 Applicant may not request that any objection to the drawing(s) be held in abeyance. See 37 CFR 1.85(a).
 Replacement drawing sheet(s) including the correction is required if the drawing(s) is objected to. See 37 CFR 1.121(d).
- 11) The oath or declaration is objected to by the Examiner. Note the attached Office Action or form PTO-152.

Priority under 35 U.S.C. § 119

- 12) Acknowledgment is made of a claim for foreign priority under 35 U.S.C. § 119(a)-(d) or (f).
- a) All b) Some * c) None of:
 1. Certified copies of the priority documents have been received.
 2. Certified copies of the priority documents have been received in Application No. _____.
 3. Copies of the certified copies of the priority documents have been received in this National Stage application from the International Bureau (PCT Rule 17.2(a)).

* See the attached detailed Office action for a list of the certified copies not received.

Attachment(s)

- | | |
|--|---|
| 1) <input type="checkbox"/> Notice of References Cited (PTO-892) | 4) <input type="checkbox"/> Interview Summary (PTO-413) |
| 2) <input type="checkbox"/> Notice of Draftsperson's Patent Drawing Review (PTO-948) | Paper No(s)/Mail Date. _____ . |
| 3) <input type="checkbox"/> Information Disclosure Statement(s) (PTO/SB/08) | 5) <input type="checkbox"/> Notice of Informal Patent Application |
| Paper No(s)/Mail Date _____ . | 6) <input type="checkbox"/> Other: _____ . |

DETAILED ACTION

The amendment filed on August 30, 2004 has been entered. Claims 1, 5-19 and 45-46 are pending, and claims 8-12, 15 and 17-19 remain withdrawn from further consideration.

Claim Rejections - 35 USC § 103

The following is a quotation of 35 U.S.C. 103(a) which forms the basis for all obviousness rejections set forth in this Office action:

(a) A patent may not be obtained though the invention is not identically disclosed or described as set forth in section 102 of this title, if the differences between the subject matter sought to be patented and the prior art are such that the subject matter as a whole would have been obvious at the time the invention was made to a person having ordinary skill in the art to which said subject matter pertains. Patentability shall not be negated by the manner in which the invention was made.

Claims 1, 5-7, 13-14, 16 and 45-46 are rejected under 35 U.S.C. 103(a) as being unpatentable over Labranque in view of Ebbeson, and further in view of Fujitani.

Labranque discloses all the claimed limitations except profiled bodies and a working pressure below atmospheric with zeolite and water.

Ebbeson (column 1, lines 13-26) discloses a sorption unit comprising zeolite bodies under vacuum and water as the working fluid for the purpose of providing heating or cooling in a refrigeration system.

Fujitani discloses a sorption unit comprising profiled bodies 7 of a sorption medium forming circular passageways for the purpose of increasing the surface absorption area to improve heat exchange efficiency.

Since Labranque, Ebbeson and Fujitani are both from the same field of endeavor and/or analogous art, the purpose disclosed by Ebbeson and Fujitani would have been recognized in the pertinent art of Labranque.

It would have been obvious at the time the invention was made to a person having ordinary skill in the art to employ in Labranque zeolite bodies under vacuum and water as the working fluid for the purpose of providing heating or cooling in a refrigeration system as recognized by Ebbeson, and employ in Labranque profiled bodies for the purpose of increasing the surface absorption area to improve heat exchange efficiency as recognized by Fujitani.

Regarding claim 14, the specific shape of the bodies is considered to be an obvious design expedient, producing no new and/or unexpected results and solving no stated problem.

Response to Arguments

Applicant's arguments have been fully considered but they are not persuasive.

The Examiner appreciates applicant's explanation of absorption versus adsorption. However, it would have been obvious to one having ordinary skill in the art at the time the invention was made to employ one "sorption medium" with another known "sorption medium", since it has been held to be within the general skill of a worker in the art to select a known material on the basis of its suitability for the intended use as a matter of obvious design choice.

In re Leshin, 125 USPQ 416. Furthermore, each sorption medium operates with a respective working fluid, but operates in the same manner, i.e. the endothermic and exothermic reactions are employed in refrigeration systems.

Conclusion

THIS ACTION IS MADE FINAL. Applicant is reminded of the extension of time policy as set forth in 37 CFR 1.136(a).

A shortened statutory period for reply to this final action is set to expire THREE MONTHS from the mailing date of this action. In the event a first reply is filed within TWO

MONTHS of the mailing date of this final action and the advisory action is not mailed until after the end of the THREE-MONTH shortened statutory period, then the shortened statutory period will expire on the date the advisory action is mailed, and any extension fee pursuant to 37 CFR 1.136(a) will be calculated from the mailing date of the advisory action. In no event, however, will the statutory period for reply expire later than SIX MONTHS from the mailing date of this final action.

Any inquiry concerning this communication or earlier communications from the examiner should be directed to Leonard R. Leo whose telephone number is (571) 272-4916. The examiner can normally be reached on Monday thru Friday.

If attempts to reach the examiner by telephone are unsuccessful, the examiner's supervisor, Cheryl Tyler can be reached on (571) 272-4834. The fax phone number for the organization where this application or proceeding is assigned is (571) 273-8300.

Information regarding the status of an application may be obtained from the Patent Application Information Retrieval (PAIR) system. Status information for published applications may be obtained from either Private PAIR or Public PAIR. Status information for unpublished applications is available through Private PAIR only. For more information about the PAIR system, see <http://pair-direct.uspto.gov>. Should you have questions on access to the Private PAIR system, contact the Electronic Business Center (EBC) at 866-217-9197 (toll-free).



LEONARD R. LEO
PRIMARY EXAMINER
ART UNIT 3744

April 30, 2007

is disturbed which thereby disrupts the boundary layer. As mentioned above, the boundary layer blankets and otherwise insulates the plate fin from the cooling fluid.

FIG. 17a shows a heat exchanger similar to that illustrated in FIG. 16, however, the plate fins have means of communication, in this case louvers 176 and slots 172. The various plate fins are joined together by support means 175. FIG. 17b shows a discrete exterior plate fin 171 having a slot 172, and FIG. 17c shows a discrete interior plate fin having louvered sections 176 and textured regions 174. The exterior plate fins 171 differ from the interior plate fins 173 in that they have no textured surface, nor do they have louvered sections 176 protruding therefrom. This is done for both aesthetics and structural reasons; the textured portions may be considered unsightly and louvers on the side plate fins may pose a risk of snagging. The benefit of this configuration is having a plate fin surface that not only disrupts the boundary layer, but also facilitates fluid communication as described above.

FIG. 18a is similar to that of 17a, however, the exterior plate fin 181 is similar to the interior plate fins. That is, both the interior plate fins and the exterior plate fins have textured regions 183 and louvered sections 182. In addition, the heat exchange has several bars 185 which transverse the plate fins. Accordingly, the alignment of the textured regions 183 and louvered regions 182 provides for fluid flow without impediments between the interior and exterior fins.

FIGS. 19a and 19b illustrate a heat exchanger incorporating a plate fin 191 similar to that illustrated in FIG. 13a. The plate fin 191 comprises perforations having rectangular shaped openings 192 with 90 degree louvered sections 193. FIG. 19a is a perspective view of a heat exchanger 190 comprised of individual plate fins 191, where both the interior and exterior fins comprise the structure as illustrated in FIG. 19b. The fins 191 of the heat exchanger are joined together by support means 195. Accordingly, the alignment of the plate fins 191 with the openings 192 causes the fluid passing along the fin to be disturbed, thereby disrupting the boundary layer and providing access to the cooling fluid.

FIG. 20a shows a heat exchanger having plate fins of varying cross-sectional area relative to the fluid flow, and FIG. 20b shows a discrete plate fin having the taper and connection means 204. In regard to the taper, plate fin 201 decreases in cross-sectional area from the intake region 202 to the exhaust region 203. Each plate fin 201 comprises connection elements 206 extending from the top surface of each plate fin 201. As the plate fins 201 are assembled, the connection elements 206 form transverse bars 207 as illustrated in FIG. 20a. Accordingly, the taper reduces the friction on the fluid moving through the fin field thereby decreasing stagnation problems.

FIG. 21a shows a variation of the plate fin shape wherein a curved notch is removed from the plate fin 211. FIG. 21b depicts a discrete plate fin 211 having a curved profile 212. The curved profile correlates to frictions exponential relationship to velocity. The function behind this particular design is to reduce the drag on fluid entering from the intake region 213, while increasing the friction on fluid entering from the exhaust region 214. Each fin comprises connection elements 216 extending from the top surface of each plate fin 211. As the plate fins 211 are assembled, the connection elements 216 align to form transverse bars 217, as illustrated in FIG. 21a. The transverse bars 217 add structural integrity to the assembled heat exchanger, as well as act as a flow control means. More specifically, the transverse bars 217 act as a flow guide means for imparting a downward force on

the fluid attempting to exit through the top surface area of the heat exchanger. Accordingly, this configuration reduces the flow of fluid in through the exhaust region.

FIG. 22a shows a heat exchanger having plate fins of varying length. The fins range from a long length 222 to a short length of 223 to from a clear region 224. This region has very little frictional effect on the fluid therefore reducing the formation of stagnant regions within the fin field. It should be clear to those skilled in the art that a variety of configurations are possible to create a clear region 224. It should also be obvious that the plate fins used in this configuration can incorporate any combination of the flow control means discussed herein. Suitable configuration include a flat plate fin 225 as shown in FIG. 22b, a slotted plate fin 226 as shown in FIG. 22c, and a perforated plate fin 227 as shown in FIG. 22d.

FIGS. 23a through 23c show a narrow channel heat exchanger 231 having a plate fin 233 with a section removed from it. As FIG. 23c shows, the removal results in a vacant area 232 which reduces the drag or friction on fluid moving through the channels. This results in reduced fluid stagnation and the problems associated therewith.

FIGS. 24a through 24c are similar to that of FIG. 23, however, rather than having an entire section removed from the plate fin, a smaller section 243 is removed from the fin 242, leaving a top portion 244 in tact. This configuration also provides for reduced drag, but also provides greater surface area for heat exchange. Furthermore, this area is located towards the top of the heat exchanger where stagnant fluid is generally not a problem.

FIGS. 25a through 25c show a narrow channel heat exchanger 251 similar to FIG. 24, however, this design also has a top plate 252 having a certain profile. The top plate 252 disturbs the fluid bypassing the fin field 251 therefore slowing it down and creating a higher pressure above the fin field. This higher pressure tends to restrict egress of fluid from the fin field 251, and may even facilitate fluid ingress.

FIG. 26a shows a heat exchanger having a plate fin made of segments 261. Segments are spaced so that a gap 263 is formed between them. In this particular embodiment, a top bar 262 is used to join the two sections together as well as connect the various plate fin components together. FIG. 26b shows a discrete plate fin component having sections 261 and gaps 263, and FIG. 26c shows a complementing plate fin arrangement having connection means 264 extending from a top surface of the plate fin. As mentioned above, when assembled in series, connection means 264 form transverse bars 262 that adds both integrity and improved heat conduction to the heat exchanger. The benefit of this design is that it allows for easy communication between the channels. Moreover, the plurality of top bars restricts the egress of fluid from the channels.

FIG. 27a shows a perspective view of a heat exchanger 271 having plate fins comprised of different components. FIG. 27b illustrates one such discrete plate fin having connection means 275 extending from the top surface of the fin. In this embodiment, the plate fin has a high surface area component 271 along with low surface area components 272. Securing the various low surface area components together are transverse bars 273, and connecting the various high surface area components together is another transverse bar 274. As shown in FIG. 27a, transverse bars 273 and 274 extend lengthwise across the heat exchanger, however bar 273 is significantly wider than bar 274. Accordingly, the benefit of this design is that it allows for easy fluid communication in the front region of the heat exchanger where fluid stagnation